

कन्वेयर चेन, चेन-पहिये और अटैचमेंट —
विशिष्टि

भाग 2 चेन-पहिये

(पहला पुनरीक्षण)

Conveyor Chains, Chain-Wheels and
Attachments — Specification

Part 2 Chains-Wheels

(First Revision)

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भारतीय मानक ब्यूरो

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FOREWORD

This Indian Standard (First Revision) was adopted by the Bureau of Indian Standards, after the draft finalized by Continuous Bulk Conveying, Elevating, Hoisting Aerial Ropeways and Related Equipment Sectional Committee had been approved by the Mechanical Engineering Division Council.

This standard (Part 2) was first published in 1976. This revision has been brought out to keep pace with the latest technological developments and the practices followed in conveyor industry and the standard has been brought into the latest style and format of Indian Standards.

This following major modifications have been incorporated in this revision of the standard:

- a) A reference clause has been added mentioning the latest version of all the referred standards;
- b) Editorial changes have been made;
- c) Marking clause has been added.

The specification for conveyor chains, chain-wheels and attachments is in three parts. This standard (Part 2) covers the chain-wheels. Other parts in this series under the general title are as follows:

Part 1 Chain

Part 3 Attachments

The composition of the Committee responsible for the formulation of this standard is given in [Annex B](#).

For the purpose of deciding whether a particular requirement of this standard is complied with, the final value, observed or calculated, expressing the result of a test or analysis, shall be rounded off in accordance with IS 2 : 2022 'Rules for rounding off numerical values (*second revision*)'. The number of significant places retained in the rounded off value should be the same as that of the specified value in this standard.

Indian Standard

CONVEYOR CHAINS, CHAIN-WHEELS AND ATTACHMENTS —
SPECIFICATION

PART 2 CHAIN-WHEELS

(First Revision)

1 SCOPE

This standard (Part 2) lays down the requirements for chain-wheels associated with chains used for conveyors.

2 REFERENCES

The standards given below contain provisions which, through reference in this text, constitute provisions of this standard. At the time of publication, the edition indicated was valid. All standards are subject to revision, and parties to agreement based on this standard are encouraged to investigate the possibility of applying the most recent edition of these standards:

IS No.	IS Title
IS 210 : 2009	Grey iron castings — Specification (<i>fifth revision</i>)
IS 2048 : 1983	Specification for parallel keys and keyways (<i>second revision</i>)
IS 919	Geometrical product specifications (GPS) — ISO code system for tolerances on linear sizes:
(Part 1) : 2014/ ISO 286-1 : 2010	Basis of tolerance, deviation and fits (<i>third revision</i>)
(Part 2) : 2014/ ISO 286-2 : 2010	Tables of standard tolerance classes and limit deviation for holes and shafts (<i>second revision</i>)
IS 6834 (Part 3) : 1976	Technical supply condition for pipe cutters (<i>first revision</i>)

3 NOMENCLATURE

It's shall be as given in [Fig. 1](#).

4 MATERIAL

The manufacturing material of the chain-wheels shall

be selected to suit design. The suitable materials are grey cast iron conforming to Grade 25 (or superior grade) of IS 210, cast steel or alloy steel with necessary heat treatment as required for the duty conditions.

5 TEETH

5.1 Chain-wheels shall be provided with teeth in accordance with [6](#) which specifies control criteria to ensure correct meshing, operating and transmission of load given under normal operating conditions. The controls do not necessarily determine the design parameters of conveyor chain-wheels.

5.2 Preferred Tooth Range

The requirements laid down in the standard apply to chain-wheels having teeth from 6 to 40. The following shall be the preferred number of teeth:

8, 10, 12, 16 and 24.

6 DIMENSIONS**6.1 Diametral Dimensions****6.1.1 Tip Diameter**

The maximum recommended tip diameter is 3 000 mm.

6.1.2 Measuring Pin Diameter

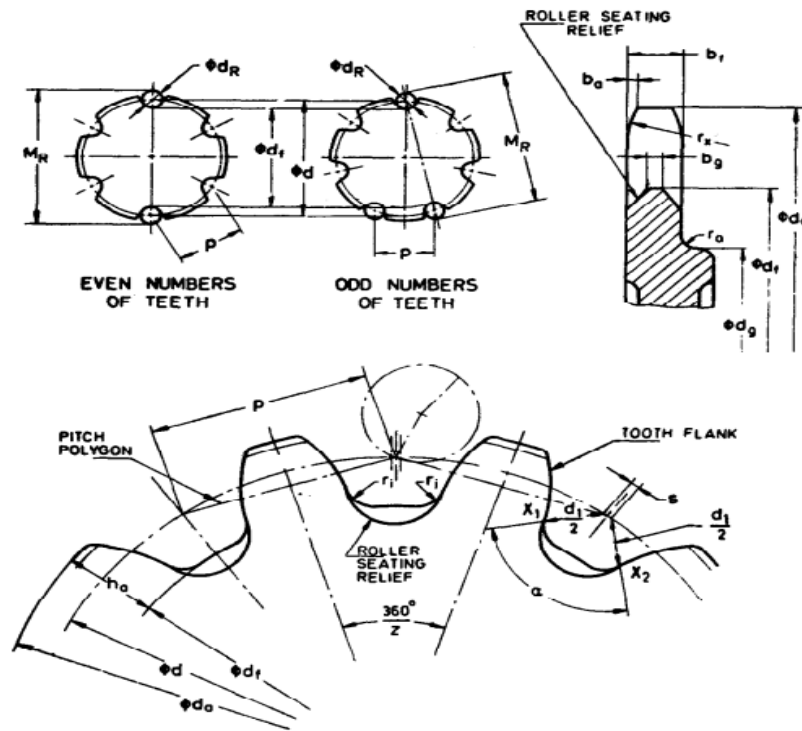
Measuring pin diameter $d_R = d_1$ (or d_4 or d_7 as appropriate) subject to tolerance $h11$ of IS 919 (Part 1) and IS 919 (Part 2).

6.1.3 Pitch Circle Diameter

Pitch circle diameter $d = p \operatorname{cosec} \frac{180^\circ}{Z}$ (see [Annex A](#) for nominal dimensions of the normal range of teeth).

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Key

- | | |
|--|---|
| b_2 = tooth side relief | h_2 = tooth height above bottom diameter |
| b_t = tooth width | |
| b_g = relieved tooth width, minimum | M = measurement over measuring pins |
| d = pitch circle diameter | P = chordal pitch and is equal to chain pitch |
| d_a = tip diameter | r_a = shroud radius |
| d_t = root diameter | r_1 = roller seating radius |
| d_g = absolute maximum shroud diameter | r_x = minimum tooth side radius |
| d_R = measuring pin diameter | S = pitch line clearance |
| d_1 = plain roller diameter, maximum | Z = number of teeth |
| d_2 = bearing pin diameter | α = roller seating angle |

FIG. 1 NOMENCLATURE FOR CHAIN-WHEELS

6.1.4 Root Diameter

Root diameter $d_{Max} = d - d_1$ (or $d - d_4$, or $d - d_7$ as appropriate) subject to limits of tolerance as follows:

Sl No.	Tooth Form	Upper Deviation	Lower Deviation
(1)	(2)	(3)	(4)
i)	As cast	0	$0.004 3 d_t + 2.6$ up to a maximum deviation of 11 mm
ii)	Machine-cut	0	$0.000 21 d_t + 0.5$ up to a maximum deviation of 0.8 mm

6.1.5 Measurement Over Measuring Pins

For even number of teeth $M_R = d + d_R$

For odd number of teeth $M_R = d \cos \frac{90^\circ}{Z} + d_R$

NOTES

1 For a wheel having an even number of teeth, measurement is made over pins inserted in diametrically opposed tooth spaces. For a wheel having an odd number of teeth, measurement is made over pins inserted in tooth spaces most nearly diametrically opposite. During measurement, the pins should always be in contact with corresponding working faces of the respective teeth.

2 The limits of tolerance for the measurement over pins are identical for those according to the corresponding root diameter.

6.2 Tooth Form and Dimensions (Cast Teeth)

The wheel tooth gap is defined taking the following criteria into consideration.

6.2.1 Working Face

The working face is the functional part of the tooth form. It is the area which lies between the-lines of contact of-two rollers, the centre line of one lying on the pitch circle and the centre line of the other lying on a circle of diameter:

$$= (p + 0.25 d_2) \operatorname{cosec} \frac{180^\circ}{Z}$$

unless reduced due to limitations imposed by having all lines inside the adjacent pitch point on the pitch circle.

The working face may be straight or convex.

6.2.2 Pressure Angle

The angle between the pitch line of the chain link and the line perpendicular to the working face at the point of roller contact. The pressure angle at any point of the working face shall be as follows:

Sl No.	Number of Teeth	Pressure Angle Degrees	
		Min	Max
(1)	(2)	(3)	(4)
i)	6 to 7	7	10
ii)	8 to 9	9	12
iii)	10 to 11	12	15
iv)	12 to 13	14	17
v)	14 to 15	16	20
vi)	16 to 19	18	22
vii)	20 to 27	20	25
viii)	28 and above	23	28

6.2.3 Tooth Height

When attachments bridge the chain link, the tip of the tooth shall not, project above the chord of pitch circle by an amount greater than 0.8 h_4 .

NOTE — h_4 = platform height of k attachments [see IS 6834 (Part 3)].

6.2.4 Minimum pitch line clearance

Minimum clearance, $s = 0.04p$ (for as cast teeth) or $0.08 d_1$ (for machine-cut teeth).

Maximum roller seating radius

$$r_1 \text{ Max} = \frac{d_1}{2} \text{ (or } \frac{d_4}{2} \text{ or } \frac{d_7}{2} \text{ as appropriate)}$$

6.2.5 Maximum Roller Seating Radius

6.2.6 Tooth Flank

6.2.6.1 Regardless of the size of the seating radius or whether a straight or curved tooth form is employed, it is essential to achieve clearance at points X_1 and X_2 of $\frac{d_1}{2}$ (or $\frac{d_4}{2}$ or $\frac{d_7}{2}$ as appropriate) between the pitch line clearance dimension lines and the tooth flank measured along the seating angle dimension lines.

6.2.6.2 Minimum clearance between tooth tip and roller track shall be $0.06 d_1$ in all cases including wheels for bush and for small roller chains.

6.3 Tooth Dimensions and Proportions (Machine-Cut Teeth)

The basic dimensions and proportions as specified in 6.1 are applicable. It may be noted that the finer tolerances than for ‘as cast’ teeth are specified against certain dimensions and these tolerances are given in the appropriate clauses of this standard.

6.4 Dimensions of Wheel Rim Profile

6.4.1 Tooth Width

For bush, small and plain roller chains:

$$b_{UMax} = 0.9 b_1 - 1 \text{ mm}$$

$$b_{UMin} = 0.87 b_1 - 1.7 \text{ mm}$$

For flanged roller chains:

$$b_{UMax} = 0.9 (b_1 - b_{11}) - 1 \text{ mm}$$

$$bt_{Min} = 0.87 (b_1 - b_{11}) - 1.7 \text{ mm}$$

6.4.2 Minimum tooth side radius— $r_x = 1.6 b_1$

6.4.3 Tooth side relief— $b_a = 0.16 b_1$

6.4.4 Minimum relieved tooth width— $b_g = 0.25 b_t$

NOTE — Under some conditions of use the material being conveyed may build up in the space between the roller and the tooth and to prevent malfunctioning it is permissible to relieve the roller seating as shown in [Fig. 1](#).

6.4.5 Shroud radius — r_a at the discretion of manufacturer.

6.4.6 Maximum shroud diameter — $d_g =$

$$P \cot \frac{180^\circ}{z} - h_2 - 2r_a$$

6.5 General Limits of Tolerance

6.5.1 Bore Diameter

Unless otherwise agreed upon between purchaser and manufacturer, bores shall conform to H9 of IS 919 (Part 1) and IS 919 (Part 2).

6.5.2 Radial Run-Out

The radial run-out between the bore and the root diameter shall not exceed a value for total indicator reading derived from:

- a) for 'as cast' teeth $0.005 d_t$ or 1.5 mm, whichever is the greater;
- b) for 'machine-cut' teeth $0.001 d_t + 0.1$ mm or 0.2 mm whichever is the greater; and

- c) Radial run-out for machine-cut teeth shall in no case exceed 2 mm.

6.5.3 Axial Run-Out

The axial run-out measured with reference to the bore and the flat part of the side face of the teeth shall not exceed the same values for total indicator readings as given in [6.5.2](#).

6.5.3.1 Axial run-out for machine-cut teeth shall in no case exceed 2 mm.

7 KEYWAYS

The provision of keyways shall be the subject of agreement between the purchaser and the manufacturer. In case keyways are provided they shall conform to IS 2048.

8 MARKING

8.1 Chain-wheels shall be marked with manufacturer's name and/or trade-mark; the number of teeth and the appropriate chain number for which the chain-wheel is suitable.

8.2 BIS Certification Marking

The product(s) conforming to the requirements of this standard may be certified as per the conformity assessment schemes under the provisions of the *Bureau of Indian Standards Act, 2016* and the Rules and Regulations framed thereunder, and the products may be marked with the Standard Mark.

ANNEX A

(Clause 6.1.3)

PITCH CIRCLE DIAMETERS

The following table gives correct pitch circle diameters for chain-wheels to suit a chain of unit pitch. The pitch circle diameters for wheels to suit a chain of any other pitch are directly proportioned to the pitch of the chain:

<i>Sl No.</i>	<i>Number of Teeth</i>	<i>Pitch Circle Diameter</i>	<i>Number of Teeth</i>	<i>Pitch Circle Diameter</i>	<i>Number of Teeth</i>	<i>Pitch Circle Diameter</i>
(1)	(2)	(3)	(4)	(5)	(6)	(7)
i)	5	1.701 3	17	5.442 2	29	9.249 1
ii)	$5\frac{1}{2}$	1.849 6	$17\frac{1}{2}$	5.600 5	$29\frac{1}{2}$	9.408 0
iii)	6	2.000 0	18	5.758 8	30	9.566 8
iv)	$6\frac{1}{2}$	2.151 9	$18\frac{1}{2}$	5.917 1	$30\frac{1}{2}$	9.725 6
—						
v)	7	2.304 8	19	6.075 5	31	9.884 5
vi)	$7\frac{1}{2}$	2.458 6	$19\frac{1}{2}$	6.234 0	$31\frac{1}{2}$	10.043 4
vii)	8	2.613 1	20	6.392 5	32	10.202 3
viii)	$8\frac{1}{2}$	2.768 2	$20\frac{1}{2}$	6.550 9	$32\frac{1}{2}$	10.361 2
—						
ix)	9	2.923 8	21	6.709 5	33	10.520 1
x)	$9\frac{1}{2}$	3.079 8	$21\frac{1}{2}$	6.868 1	$33\frac{1}{2}$	10.679 0
xi)	10	3.236 1	22	7.026 6	34	10.838 0
xii)	$10\frac{1}{2}$	3.392 7	$22\frac{1}{2}$	7.185 3	$34\frac{1}{2}$	10.996 9
—						
xiii)	11	3.549 4	23	7.343 9	35	11.155 8
xiv)	$11\frac{1}{2}$	3.706 5	$23\frac{1}{2}$	7.502 6	$35\frac{1}{2}$	11.314 8
xv)	12	3.863 7	24	7.661 3	36	11.473 7
xvi)	$12\frac{1}{2}$	4.021 1	$24\frac{1}{2}$	7.820 0	$36\frac{1}{2}$	11.632 7
—						
xvii)	13	4.178 6	25	7.978 7	37	11.791 6
xviii)	$13\frac{1}{2}$	4.336 2	$25\frac{1}{2}$	8.137 5	$37\frac{1}{2}$	11.950 6
xix)	14	4.494 0	26	8.296 2	38	12.109 5
xx)	$14\frac{1}{2}$	4.651 8	$26\frac{1}{2}$	8.455 0	$38\frac{1}{2}$	12.268 5
—						
xxi)	15	4.809 7	27	8.613 8	39	12.427 5
xxii)	$15\frac{1}{2}$	4.967 7	$27\frac{1}{2}$	8.772 6	$39\frac{1}{2}$	12.586 5
xxiii)	16	5.125 8	28	8.931 4	40	12.745 5
xxiv)	$16\frac{1}{2}$	5.284 0	$28\frac{1}{2}$	9.090 2	—	—

ANNEX B

(Foreword)

COMMITTEE COMPOSITION

Continuous Bulk Conveying, Elevating, Hoisting Aerial Ropeways and Related Equipment Sectional Committee, MED 06

<i>Organization</i>	<i>Representative (s)</i>
Rites Limited, Gurugram	SHRI R. K. SHARMA (<i>Chairperson</i>)
Adventure Tour Operator Association, New Delhi	SHRI AJEET BAJAJ SHRI PRADEEP MURTHY (<i>Alternate I</i>) SHRI NIRAT BHATT (<i>Alternate II</i>)
Bharat Heavy Electricals Limited, Project Engineering Management, Noida	SHRI PANKAJ KAPSIMAY SHRI VIVEK HEMROM (<i>Alternate</i>)
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Damodar Ropeways & Infra Limited, Kolkata	SHRI D. L. DAS
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Directorate General of Mines Safety, Dhanbad	SHRI D. B. NAYAK SHRI VIJAY YADAV RAO BARAPATRE (<i>Alternate</i>)
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National Highways Logistics Management Limited, New Delhi	SHRI SUNIL YADAV
National Institute of Technology, Kurukshetra	SHRI MANOJ KUMAR GUPTA SHRI VIKAS KUMAR
Ropeway and Resorts Pvt Ltd, Kolkata	SHRI BIPLAB DAS SHRI SUDIPTA KRISHANA MANDAL (<i>Alternate</i>)

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